

Fluid Arbeitssysteme

Literaturempfehlung:

Christopher Brennen Caltech

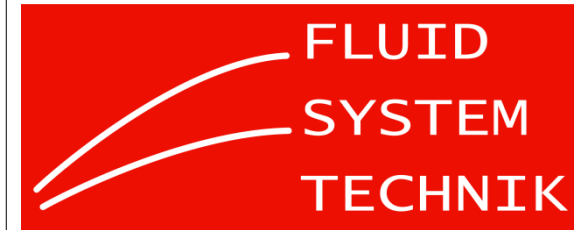
Hydrodynamics of Pumps

* PDF frei verfügbar.

* Link



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Die Aufgabe eines Fluidesystems
Förderaufgabe für

- * Wauw
- * chemische Analyse
- * Kühlungsaufgabe
- ⋮

} \dot{V}_{Soll} !

Aufgabe besteht darin eine Sollvolumenstrom
möglichst effizient (geringer Energieverbrauch)
einzustellen.

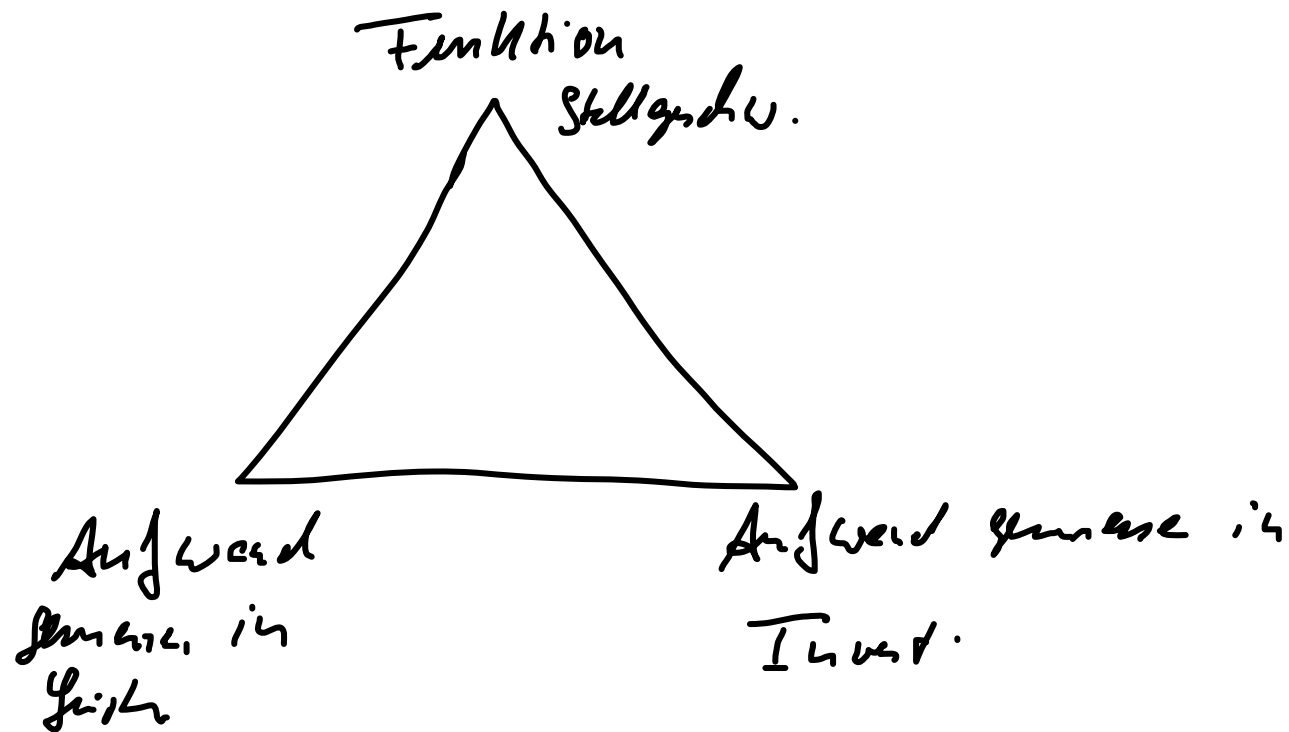


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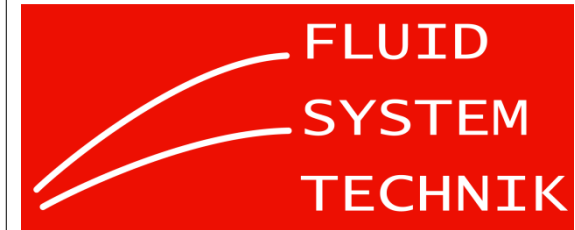
1. Anforderung an die Effizienz

2. " an die Stellgeschwindigkeit.

3. " geringe Turbulenzverluste.

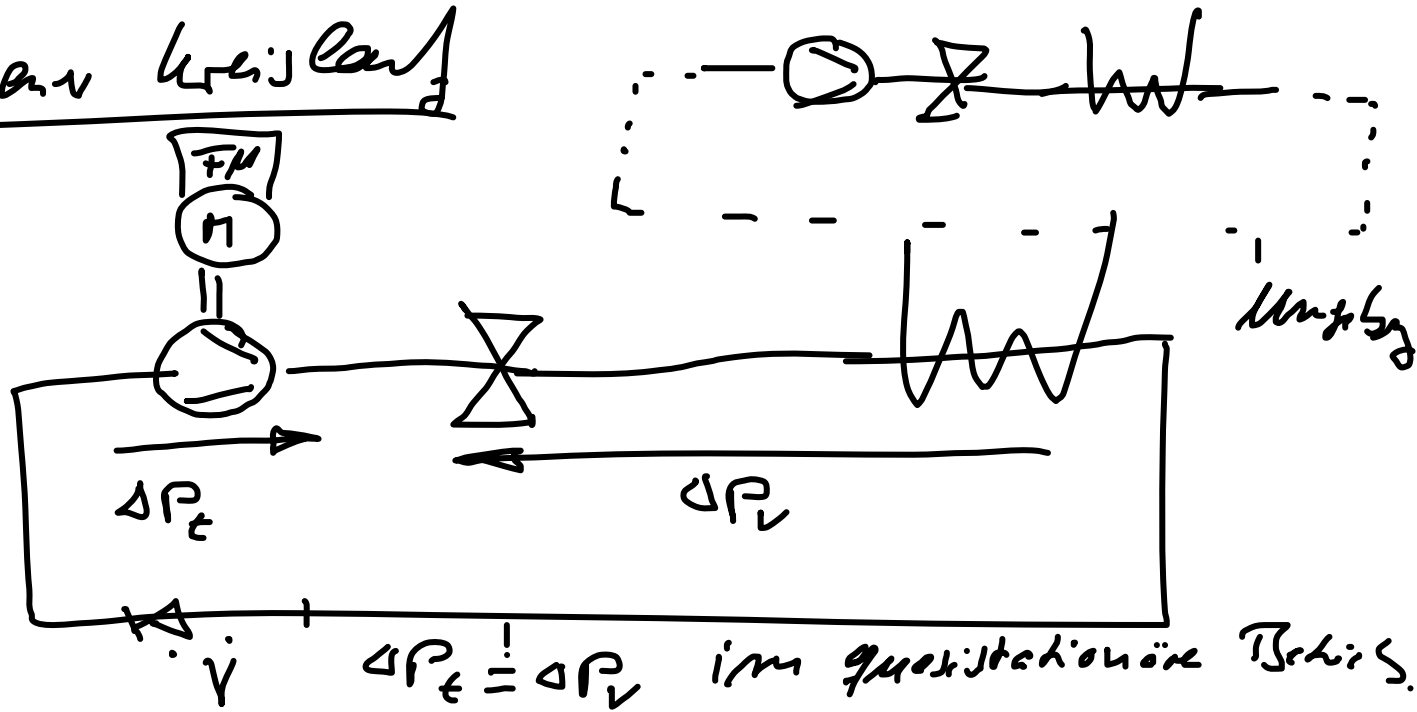


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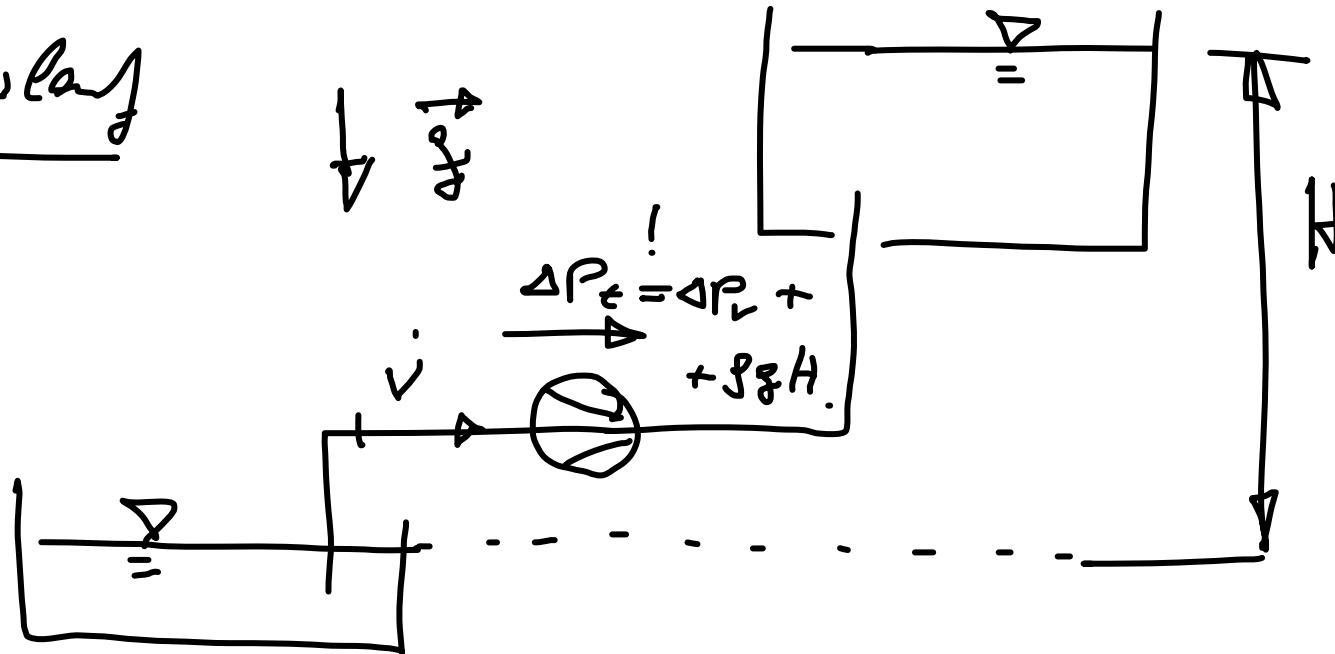


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geschlossener Kreislauf



offener Kreislauf

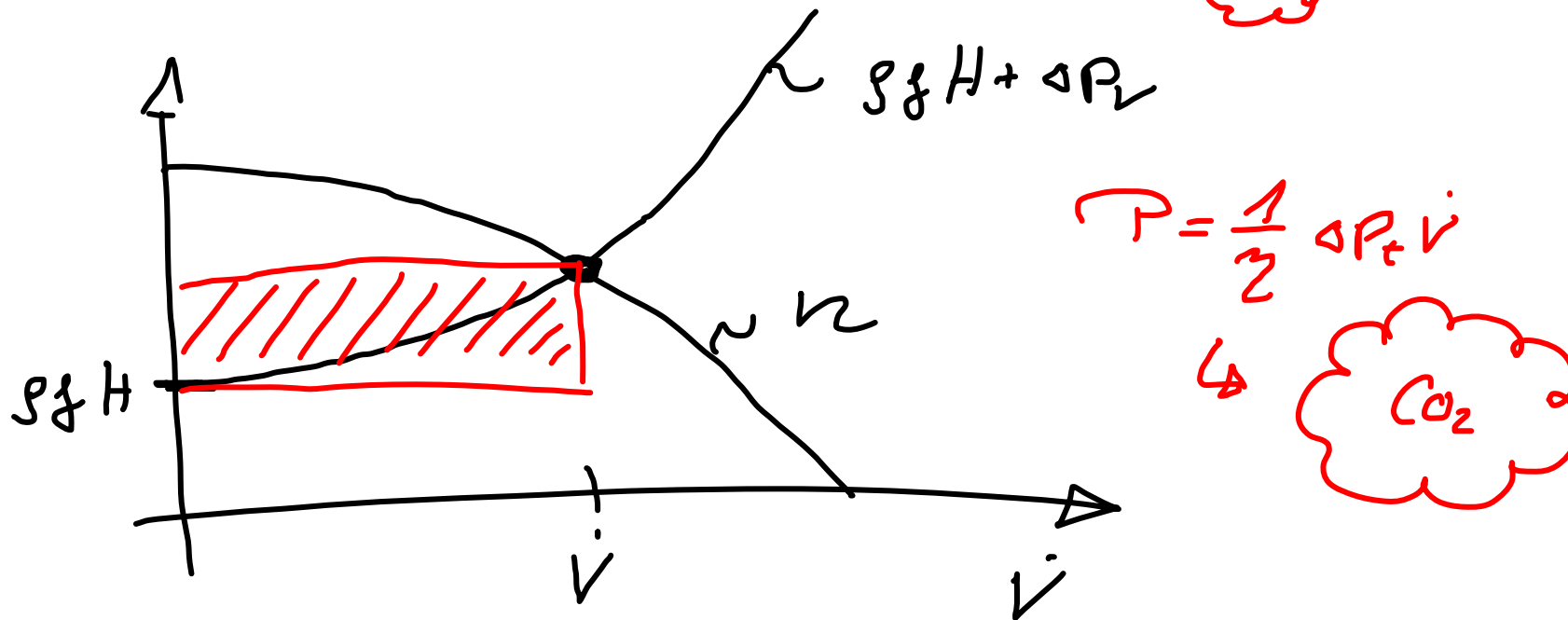
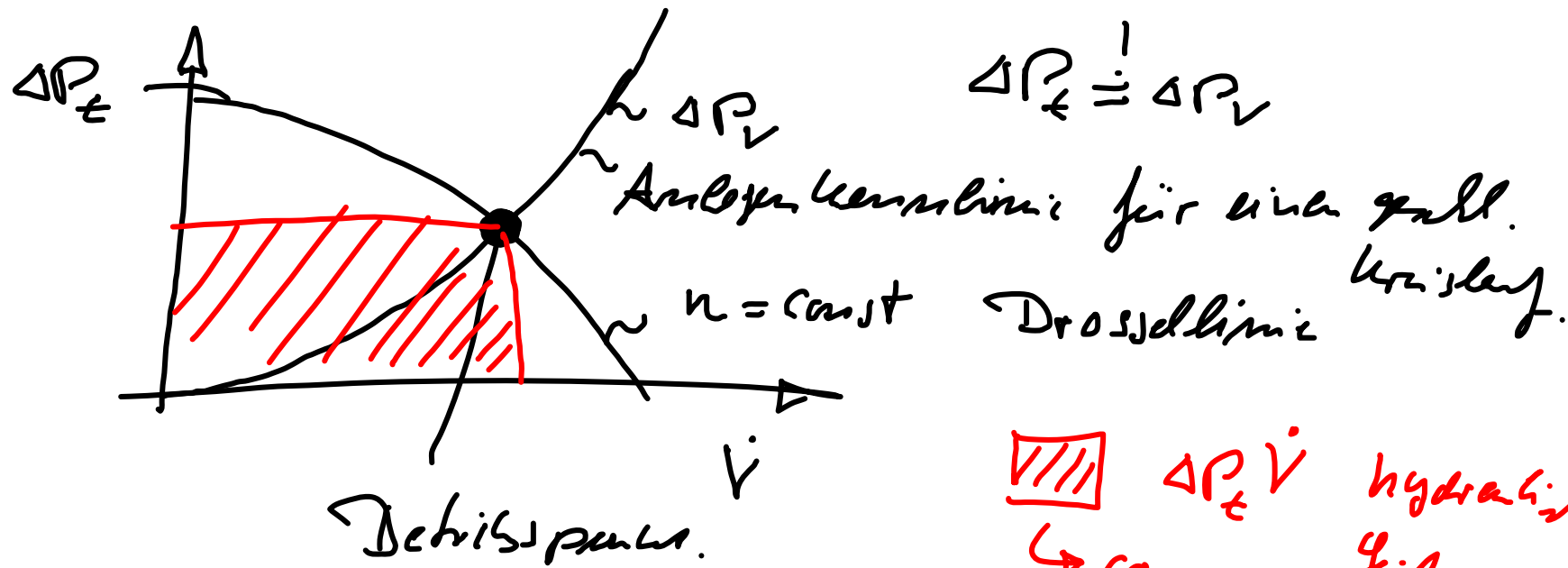


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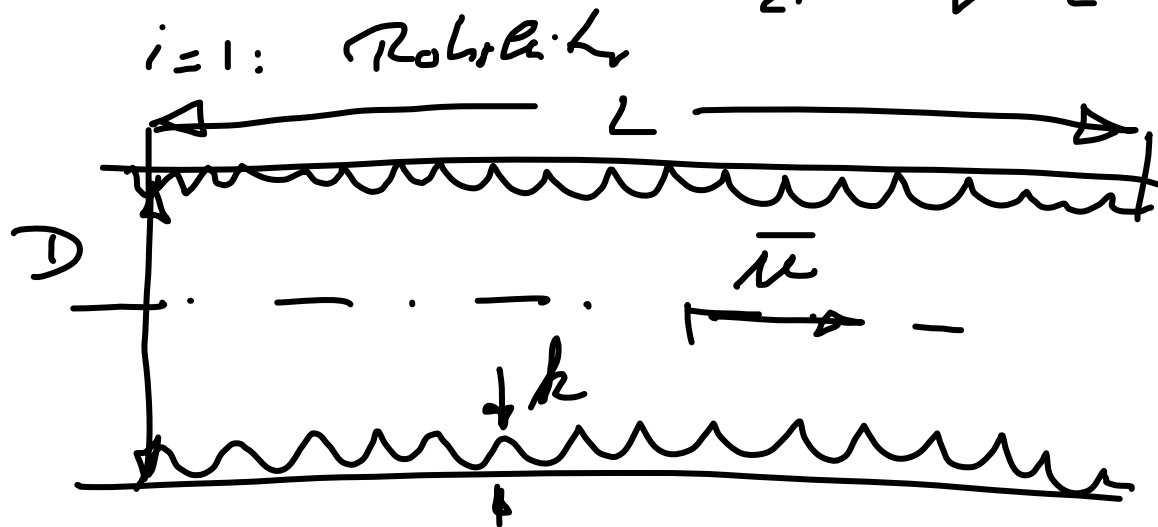
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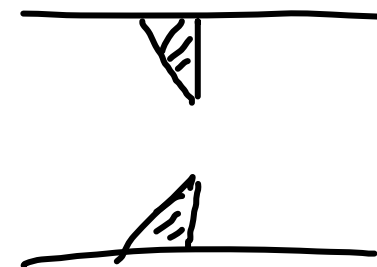
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$$\Delta P_V = \sum_{i=1}^n \Delta P_{V_i} = \frac{\rho \bar{u}^2}{2} \frac{L}{D} \lambda \left(\frac{\bar{u} D}{\nu}, \frac{k}{D} \right) + \frac{\rho \bar{u}^2}{2} J_V + \frac{\rho \bar{u}^2}{2} J_E + \frac{\rho \bar{u}^2}{2} J_A + \dots$$



$i=2$: Ventil



$i=3$: Elkhaken

$i=4$ Anschlüsse!

$i=5$ Filz.

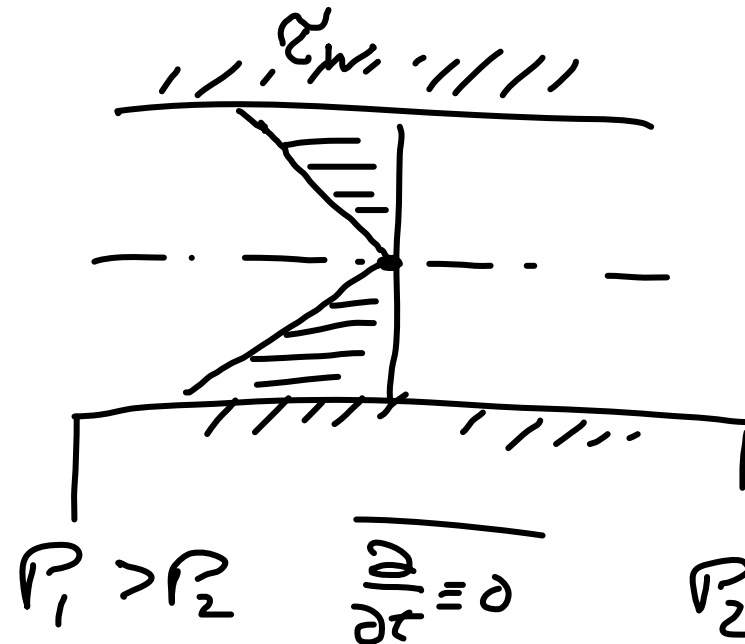


$$\lambda (Re, k/D)$$

||
 $\frac{\bar{\mu} D}{\nu}$ relative
Rauheit.

Drehmoment pro
Längeneinheit $\hat{=} \text{dimensionlos}$.
Wand Schubspannung.

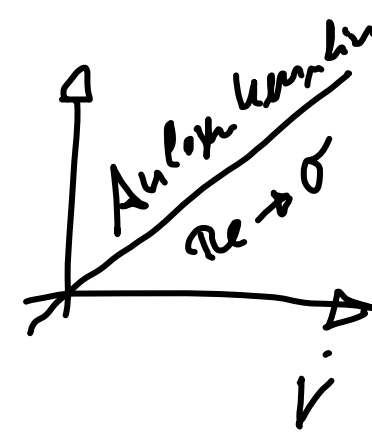
$$\lambda = \frac{\tau_w}{\frac{\rho}{2} \bar{u}^2}$$



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$$\lim_{Re \rightarrow 0} \lambda(Re, k_+) = \text{const} \frac{1}{Re}$$



da im Grenzfall rollende Strömung
die Wandschubspannung $\tau_w \neq f_w(\rho)$

$$\nabla \quad \frac{\tau_w}{\frac{1}{2} \rho \bar{v}^2} = \text{const} \frac{\eta}{\bar{\mu} D} \quad \underline{\text{Viskose Durchdringung}}$$

$$\leadsto \tau_w = \text{const} \frac{\eta}{D} \bar{v} \quad ; \quad \bar{\mu} = \frac{\eta}{\frac{1}{4} \pi D^2}$$

$$\Delta p_v \sim \tau_w \sim \dot{v}$$



$$\lim_{Re \rightarrow \infty} \lambda(Re, k_+) = \lambda(k_+)$$

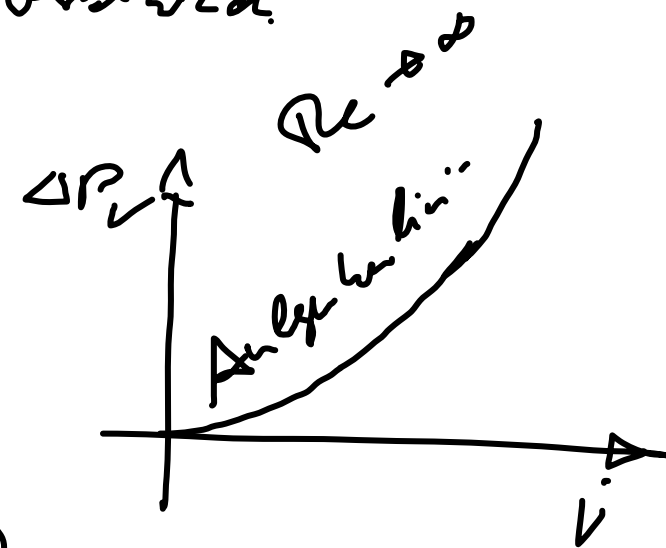
Im Grenzfalle $Re \rightarrow \infty$ darf die Vordruckspannung nicht von der Viskosität η od. ν abhänge.

\leadsto Die Reynoldszahl muß verbleiben.

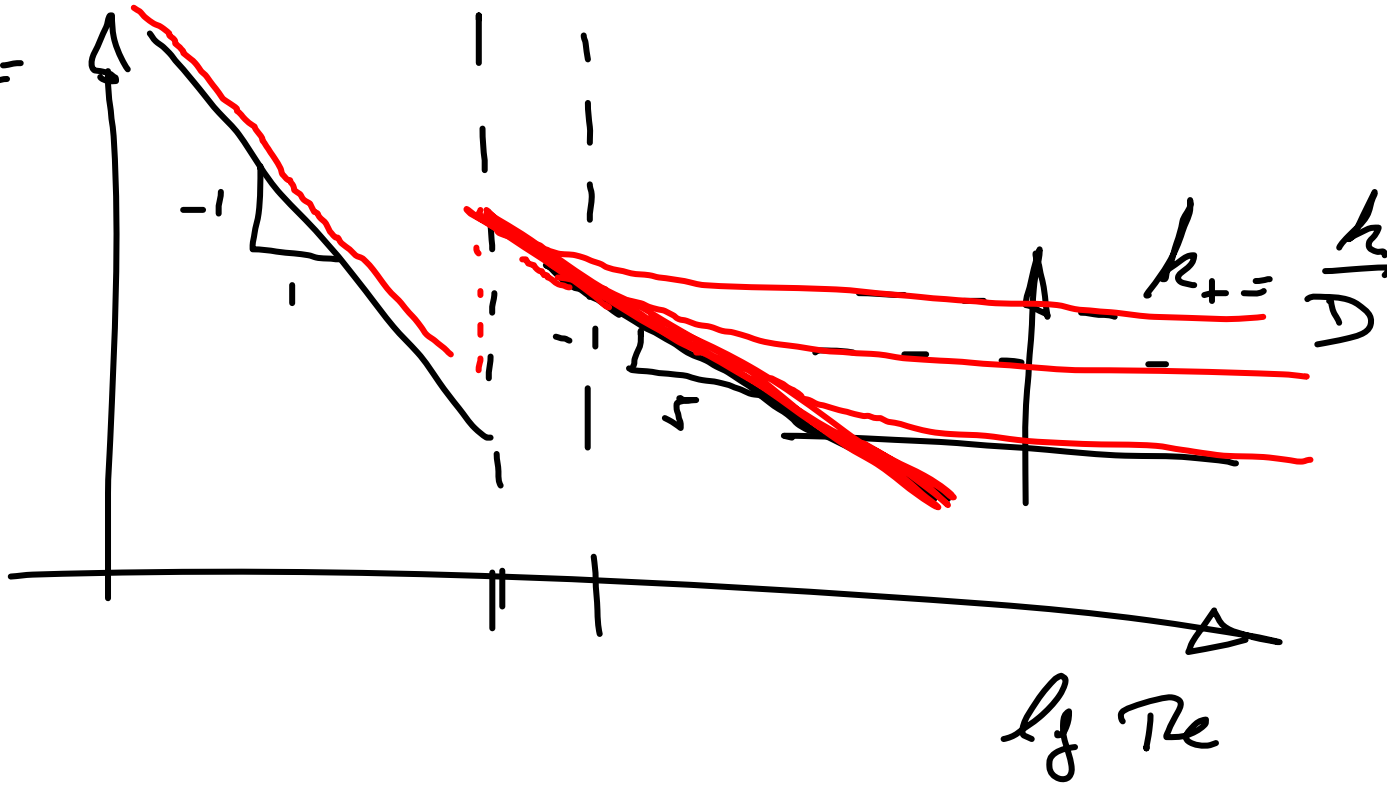
$$\frac{\tau_w}{\frac{\rho}{2} \bar{v}^2} = \lambda(k_+)$$

$$\tau_w = \frac{\rho}{2} \bar{v}^2 \lambda(k_+)$$

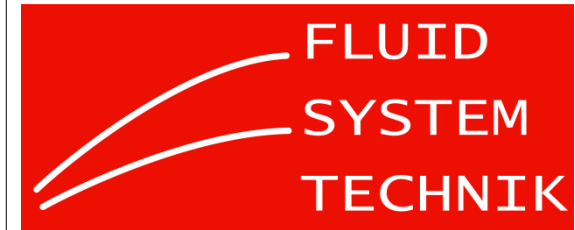
$$\Delta P_v \sim \bar{v}^2 \quad \text{für festes Durchmess.}$$



$f \lambda =$
 $\frac{2 \nu}{v} \frac{v}{D}$



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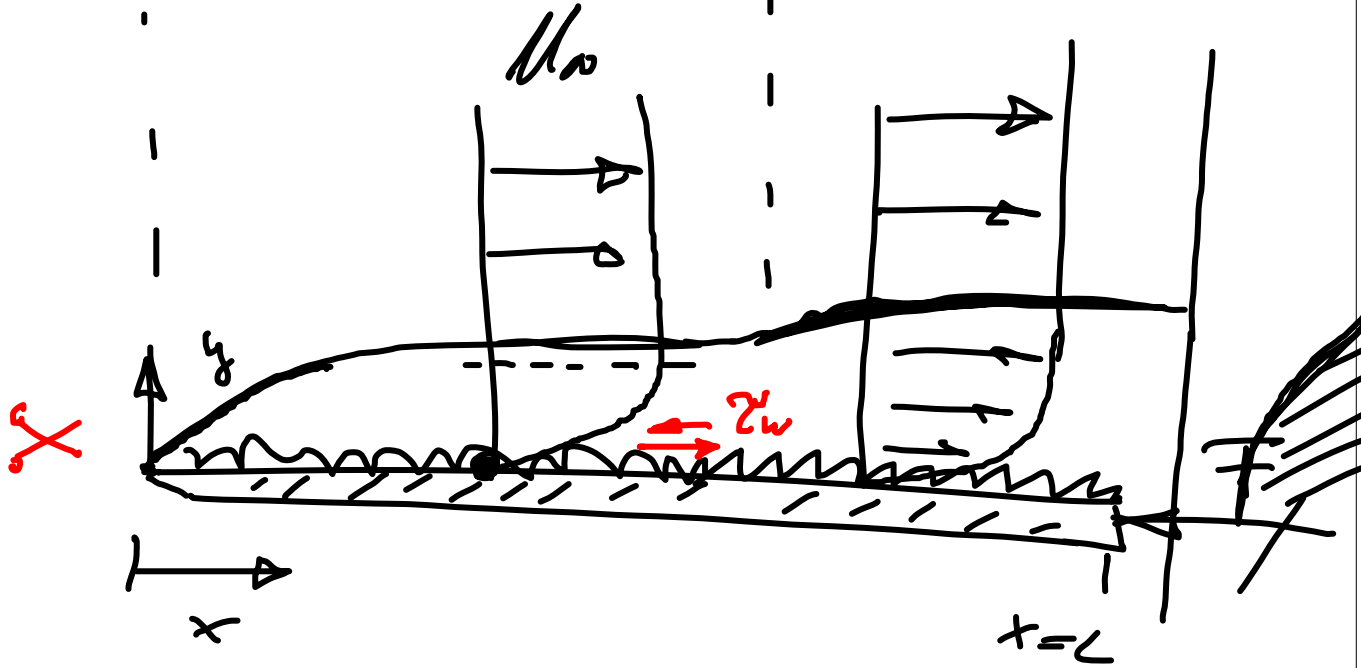


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Platte

laminare Grenzschicht

turbulente Grenzschicht



$$F = \int_0^L \tau_w dx$$

$$\frac{F}{\frac{\rho}{2} M_\infty^2 L} = C_w \left(\frac{M_\infty L}{\nu}, \frac{h}{L} \right)$$

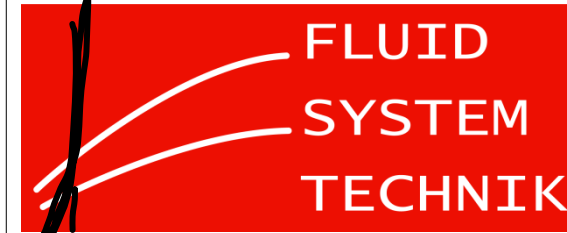
Impulsverlust in jeder Grenzschicht

~>

Widerstandskraft \$F\$ pro Tiefeneinheit



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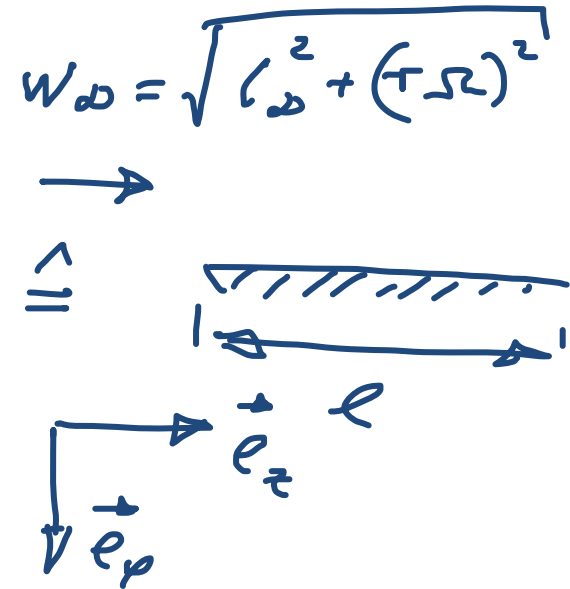
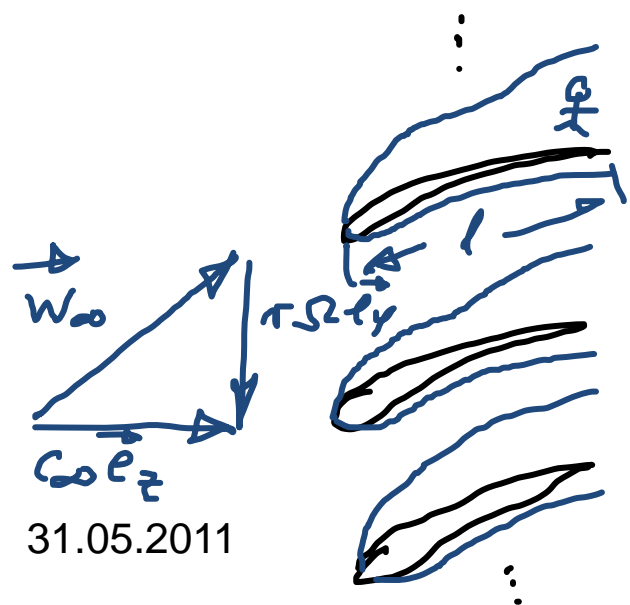
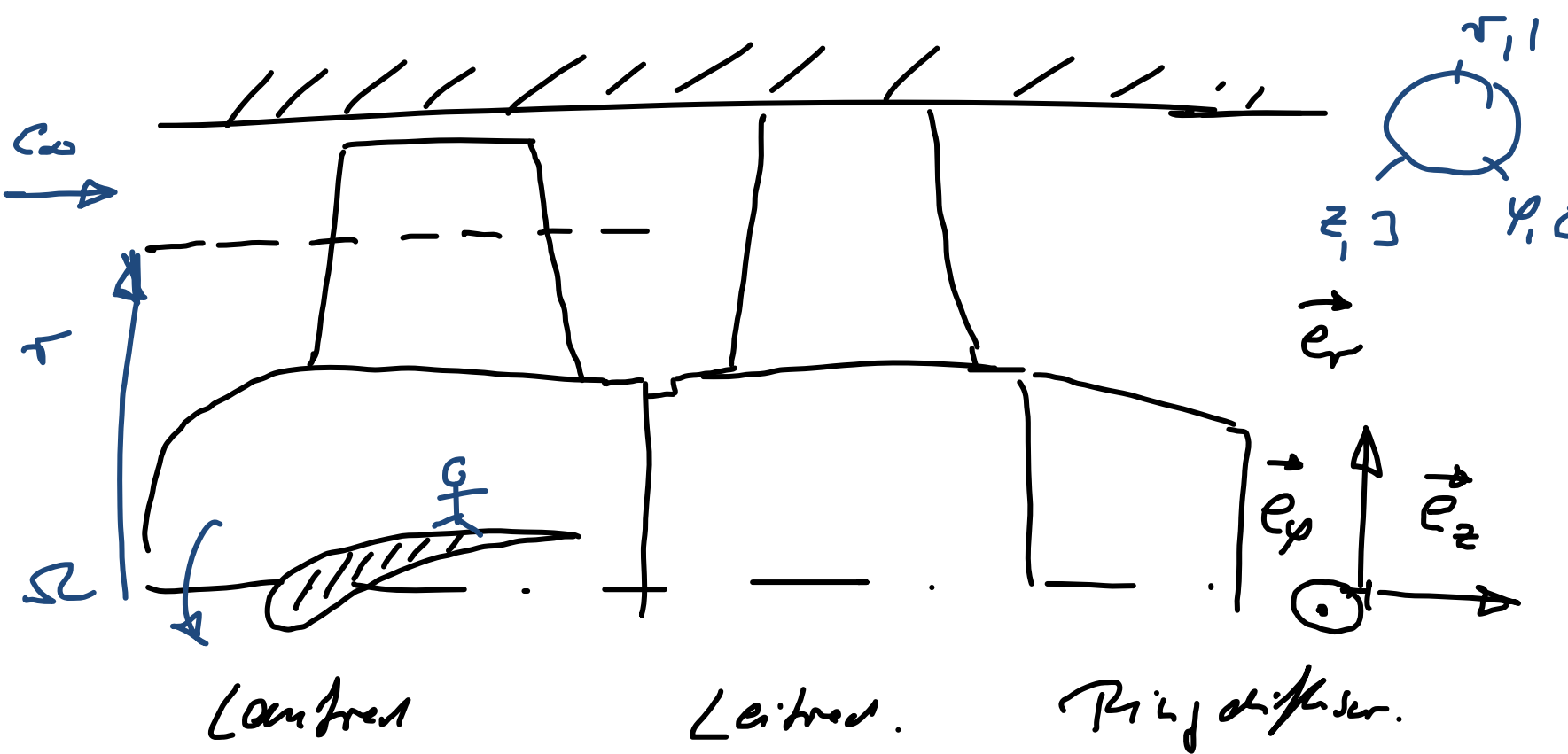
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$$\frac{\tau_w}{\frac{\rho}{2} u_\infty} := c_f \left(\frac{x}{L}, \frac{u_\infty L}{\nu}, \frac{h}{L} \right)$$

Reibbeiwert.



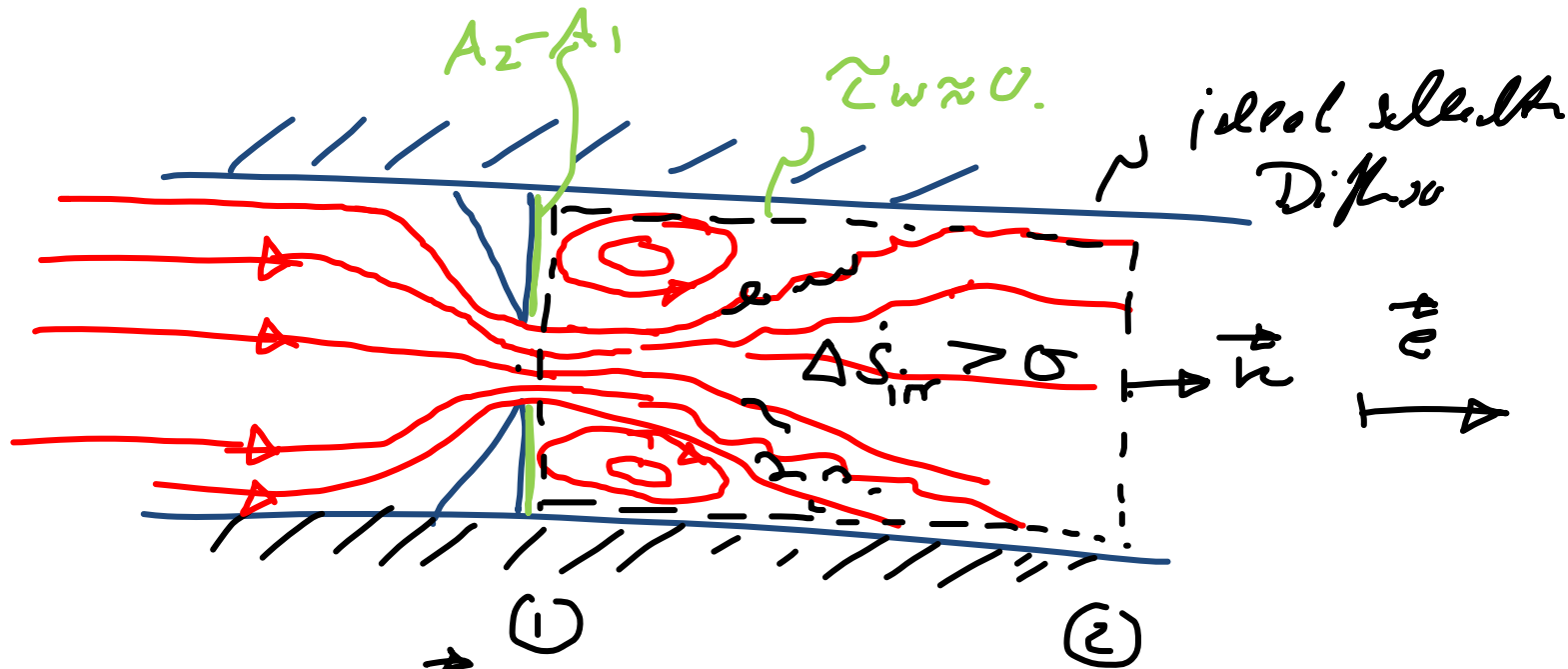
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Ventilströmung: Trägheitsänderung v.



1. Impulsbilanz \vec{e}

$$-\rho M_1^2 A_1 + \rho M_2^2 A_2 = P_1 A_1 - P_2 A_2 - \underbrace{F}_{Fl \rightarrow Wand.}$$

2. Bernoulli: $\leadsto (P_2 - P_1)_{\text{real}} = \dots$

$$- (A_2 - A_1) P_1$$

$$P_1 + \frac{\rho}{2} M_1^2 = P_{2 \text{ ideal}} + \frac{\rho}{2} M_2^2 \leadsto (P_2 - P_1)_{\text{ideal}}$$



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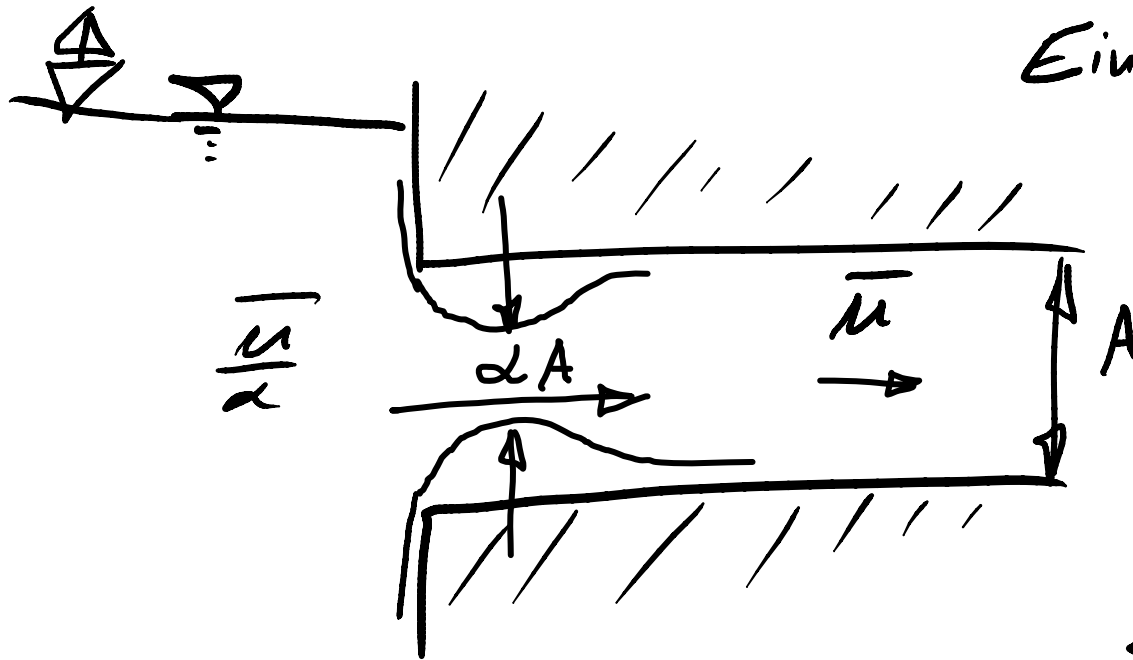
$$\Delta P_L = (P_2 - P_1)_{\text{ideal}} - (P_2 - P_1)_{\text{real}}$$

$$= \frac{\rho}{2} (u_1 - u_2)^2 \quad \text{Carnotscher Stoßverlust.}$$

~~dissipiert geht~~

$$P_L = \Delta P_L u_1 A \neq \int L(z)$$

und Treiberleistung.



Eintrittsverlust.

$$\frac{\Delta P_v}{\frac{\rho}{2} \bar{u}^2} = \left(\frac{\alpha - 1}{\alpha} \right)^2$$

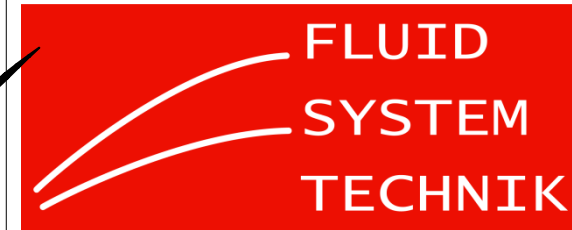
$$\Delta P_v = \frac{\rho}{2} \bar{u}^2 \left(\frac{1}{\alpha} - 1 \right)^2$$

α Kehrwertverhältnis

$\alpha \leq 1$



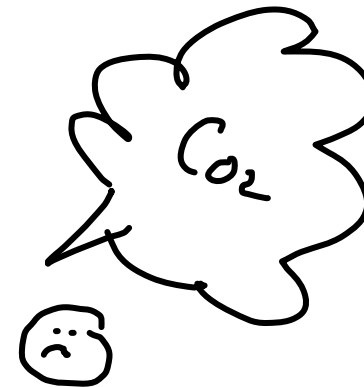
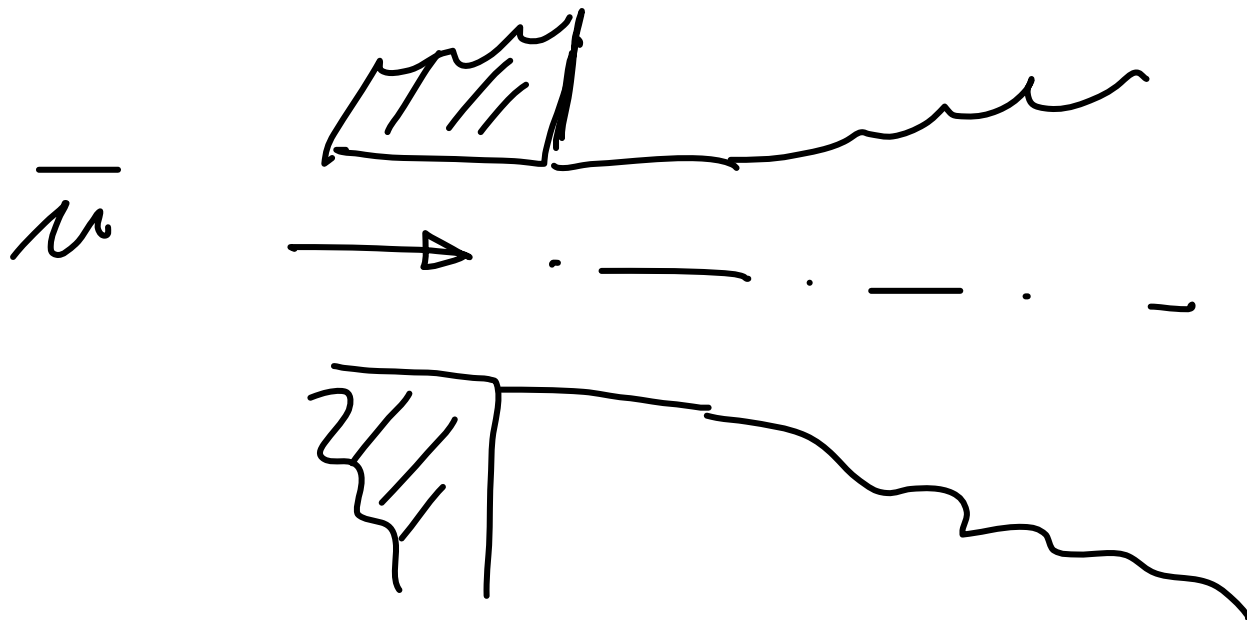
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Austrittsdruckverlust $M_2 \ll M_1$ $\frac{A_2}{A_1} \rightarrow \infty$

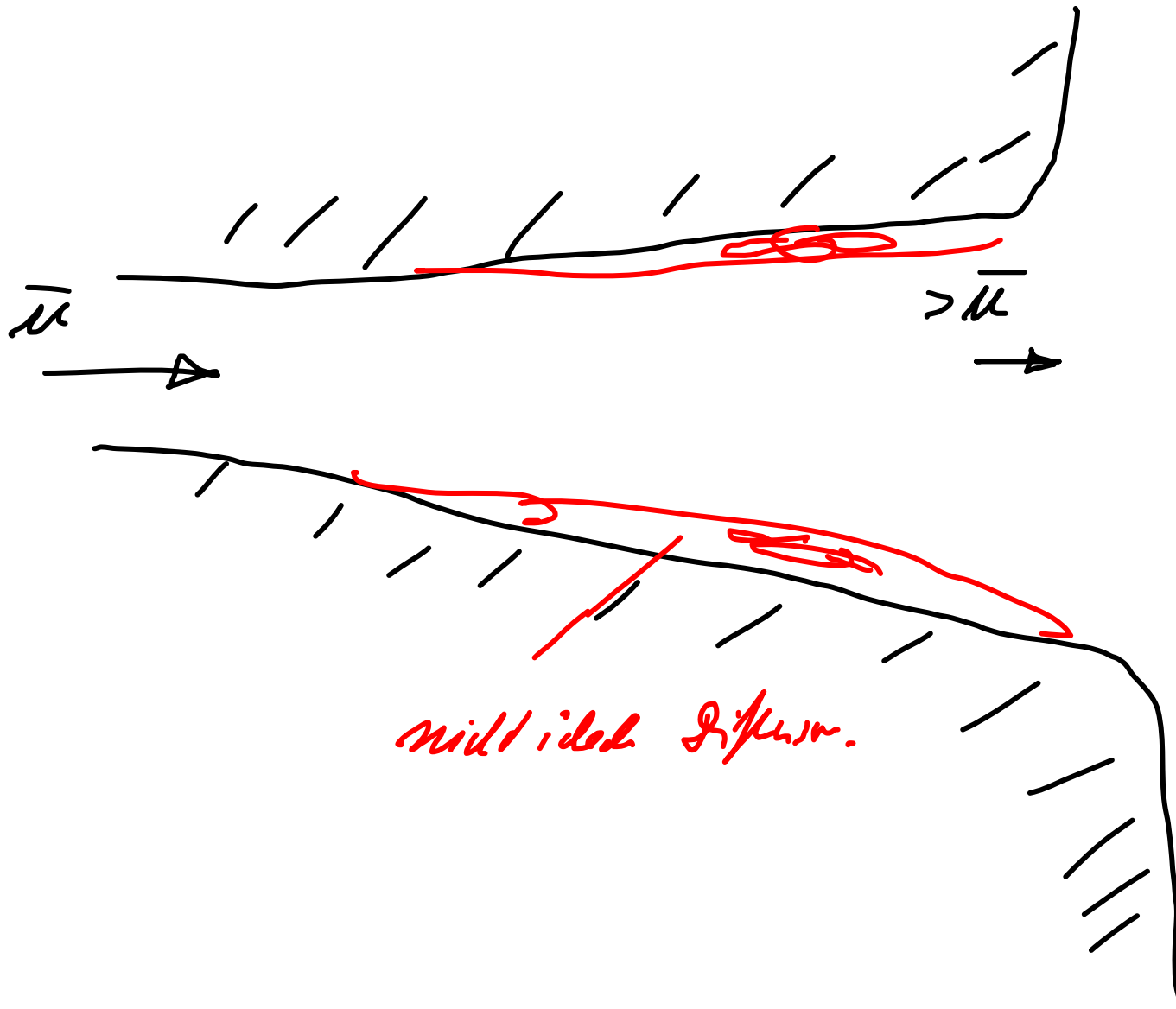
$$\Delta P_v = \frac{\rho}{2} M_1^2 = \frac{\rho}{2} \bar{u}^2$$

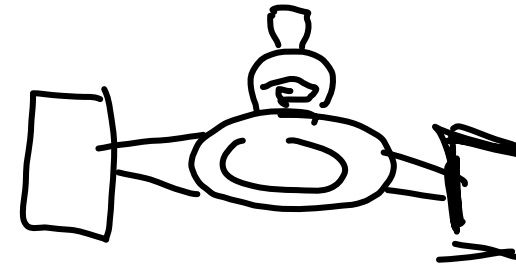
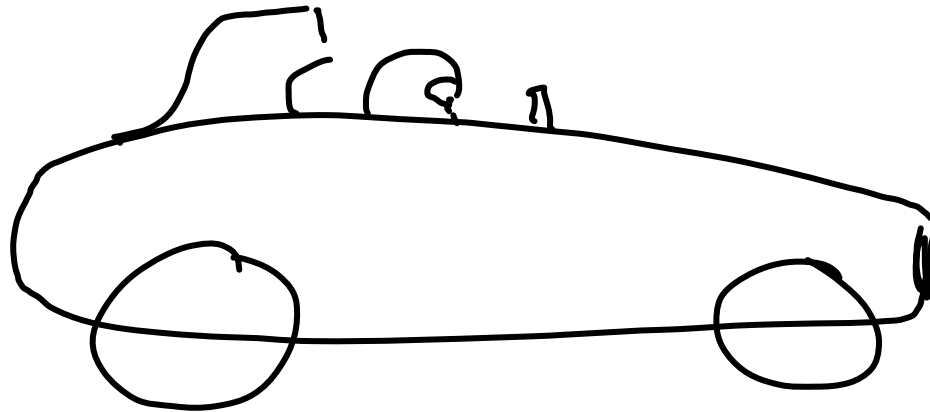


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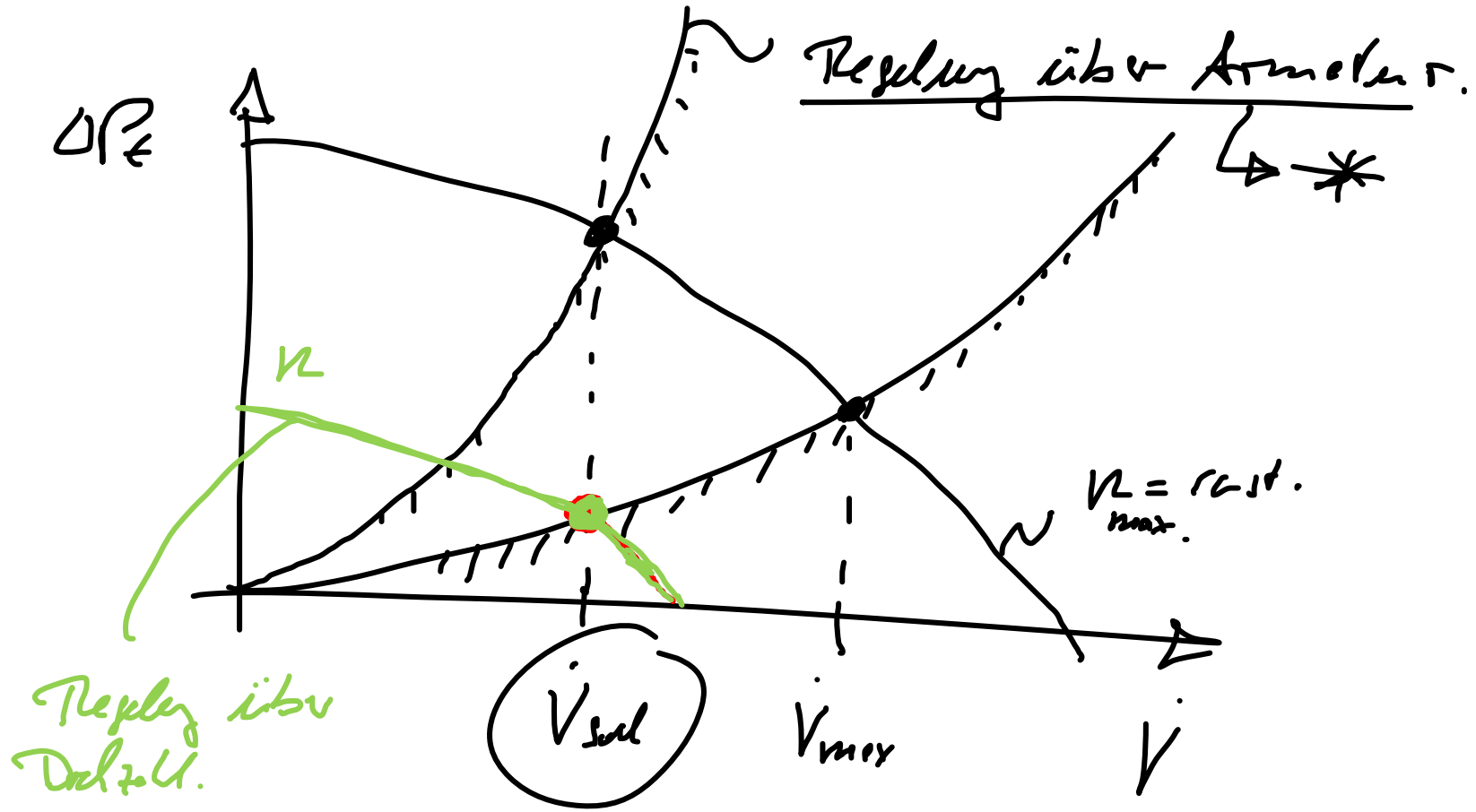




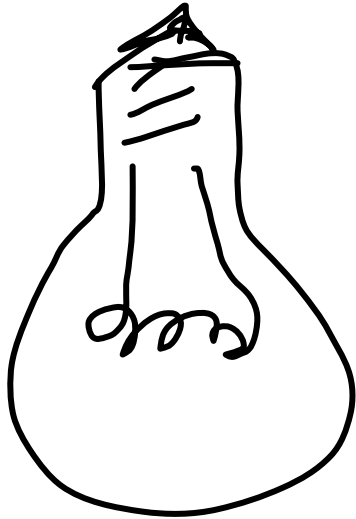
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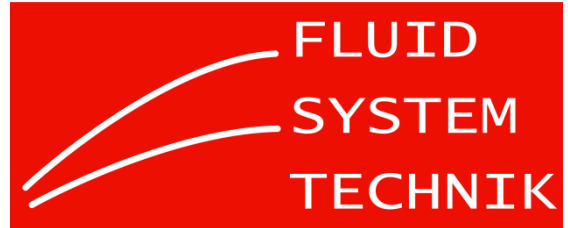
* Aktuatorik in der Prozessautomatisierung
Verfahrenstechn. Anl. Dr. Kirsbacher
Vorstand Fa. Samson.



ErP-Richtlinie Energy related Products.



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